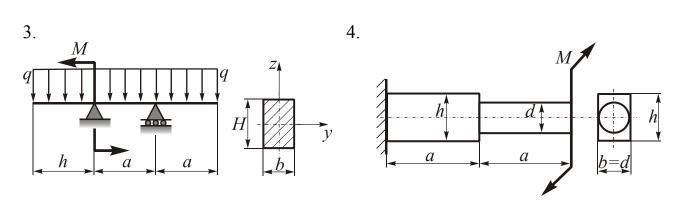
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 1

- 1. Parallel transfer of axes of inertia. Calculation of axial moments of inertia in parallel transfer of axes (proof).
- 2. Mechanical properties of structural materials and calculation of allowable stresses under the results of mechanical testing.



- <u>Given:</u> $a=3 m, q=5 kN/m, M=6 kNm, [\sigma]_c=200 MPa, [\sigma]_t=100 MPa, h/b=2.$
- **<u>Aim:</u>** find dimensions of rectangular crosssection and draw the graphs $\sigma_x(z)$ and $\tau_{xz}(z)$ in critical section.

<u>Given:</u> h=1.5b, $b=d=4\times 10^{-2}$ m, [τ] =80 MPa.

<u>Aim:</u> determine allowable value of external torsional moment [M].

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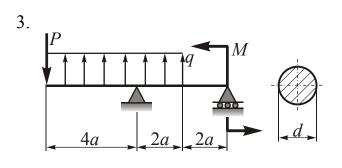
Examiner

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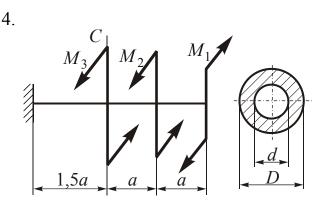
Examination card № 2

- 1. Rotation of axes of inertia. Calculation of principal axes position. Principal moments of inertia (proof).
- 2. Relationship between internal forces and stresses.



<u>Given:</u> a=0.5 m, q=P/l, l=2 m, M=6Pl, $[\sigma]=100 MPa, d=2 \times 10^{-2} m.$

<u>Aim:</u> determine [P].



- <u>Given:</u> a=0.3 m, $\alpha=d/D=0.6$, $G=8\times10^4 MPa$, $[\theta]=1 \ degree/m$, $M_1=8 \ kNm$, $M_2=20 \ kNm$, $M_3=10 \ kNm$.
- <u>Aim:</u> determine diameters d and D of the shaft and angle of twist of C-section

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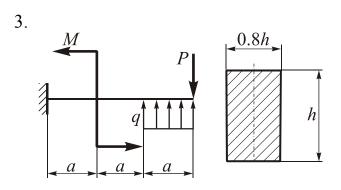
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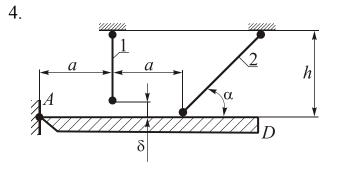
Examination card № 3

- 1. Stress state at a point of deformable solid. Formulae for stresses acting on two mutually perpendicular planes of general position in plane stress state (proof).
- 2. Factors of material ductility (plasticity) in tension.



<u>Given:</u> a=1 m, q=2 kN/m, P=6 kN, M=4 kNm, $[\sigma]_t=[\sigma]_c=200 MPa$.

<u>Aim</u>: determine h.



<u>Given:</u> AD - absolutely rigid bar, $a=h=1.0 m, \alpha=30^{\circ}, A_1=A_2=A=1\times10^{-4}$ $m^2,$ $E_1=E_2=2\times10^5 MPa, \quad \delta=5\times10^{-4} m.$

<u>Aim:</u> calculate stresses in rods *l* and *2* after assembly.

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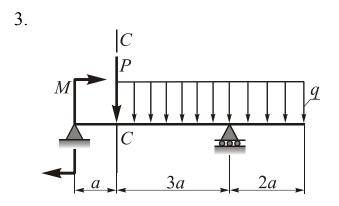
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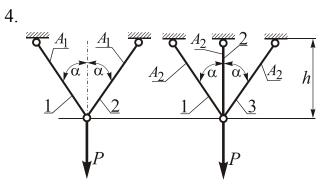
Examination card № 4

- 1. Plane stress state. Proof of the formulae for principal stresses and principal planes position.
- 2. Principle of superposition.



<u>Given:</u> a=1 m, q=5 kN/m, P=12 kN, $M=6 kNm, [\sigma]_c=200 MPa,$ $[\sigma]_t=150 MPa$

<u>Aim</u>: determine the number of I-beam section and calculate τ_{max} in *C*-section.



<u>Given:</u> h=1 m, P=20kN, $\alpha=30^{\circ}$, $[\sigma]=160 MPa$, $\rho=7.8 \times 10^{3} kg/m^{3}$.

<u>Aim</u>: find A_1 , A_2 for two variants of rod system and compare their masses.

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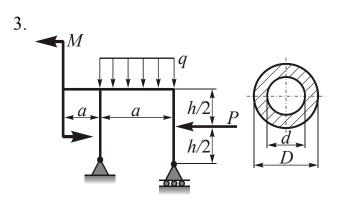
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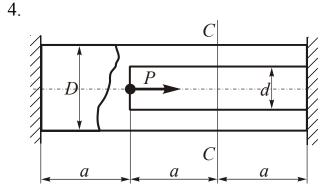
Examination card № 5

- 1. Graphical solution of direct and inverse problems of plane stress state (Mohr's circles).
- 2. External and internal forces, their types, units. Difference between external and internal forces. Internal stress as the measure of internal force distribution in cross-section.



<u>Given:</u> a=1 m, h=2 m, q=2 kN/m, $P=4 kN, M=8 kNm, [\sigma]_c=200 MPa,$ $[\sigma]_t=100 MPa, d/D=0.8.$

<u>Aim:</u> draw graphs N_x , Q_z , M_y ; calculate D and d for the most loaded crosssection.



<u>Given:</u> $a=1.0 m, P=60 kN, D=12 \times 10^{-2} m, d=6 \times 10^{-2} m, E=2 \times 10^{5} MPa.$

<u>Aim:</u> draw graph $N_x(x)$; determine displacement of C-section.

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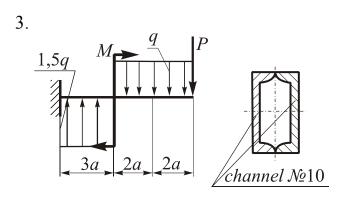
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 6

1. Generalized Hooke's law for principal planes (proof).

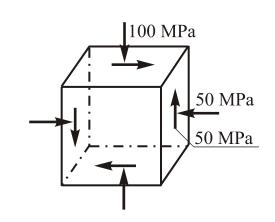
2. Rational orientation of cross-section relative to plane of loading in plane bending (examples).

4.



<u>Given:</u> $a=2 m, P=2qa, M=4qa^2, [\sigma]_c=200 MPa, [\sigma]_t=150 MPa.$

Aim: determine [q].



<u>Given:</u> $E=2\times 10^5$ MPa, $\mu=0,3$.

<u>Aim:</u> find position of principal planes.Calculate principal stresses, maximum shear stresses and also relative

change in volume ε_{v} .

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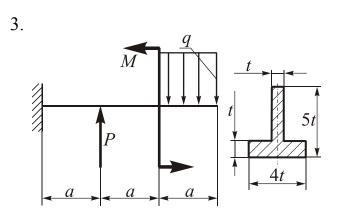
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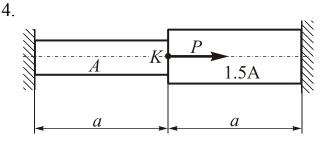
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 7

- 1. Strain state at a point of elastic deformable solid. Relationship between components of strain state and displacements *U*, *V*, *W* at a point (Cauchy relationships)
- 2. Sectional moduli of cross-sections (examples of calculation).



- <u>Given:</u> a=1 m, M=30 kNm, q=40 kN/m, P=20 kN, $t=4 \times 10^{-2} m$, $[\sigma]_t=100 MPa$, $[\sigma]_c=200 MPa$.
- <u>Aim</u>: calculate σ_{max} and check the strength of the beam.



- **<u>Given:</u>** $a=1 m, P=80 kN, [\sigma]_c=160 MPa, [\sigma]_t=100 MPa.$
- <u>Aim:</u> calculate cross-sectional area A, and also displacement of point K.

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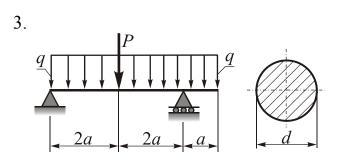
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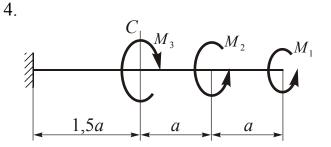
Examination card № 8

- 1. Tension-compression of the rod. Hypothesis of plane sections. Formula for normal stresses (proof). Condition of strength.
- 2. Axial and polar moments of inertia and also product of inertia for plane figures. Relationship between axial and polar moments of inertia.



<u>Given:</u> $a=1 m, P=3qa, d=8 \times 10^{-2} m, [\sigma]_t=180 MPa, [\sigma]_c=200 MPa.$

<u>Aim:</u> draw graphs $Q_z(x)$, $M_y(x)$ and determine allowable intensity of distributed load [q].



- **<u>Given:</u>** $[\tau] = 100 MPa, G = 8 \times 10^4 MPa,$ $[\psi] = 1 degree/m, a = 1.0 m, d/D = 0.8,$ $M_1 = 1.5 kNm, M_2 = 4.5 kNm,$ $M_3 = 3 kNm,$
- <u>Aim:</u> draw graph $M_x(x)$, calculate diameters of hollow shaft and also angle of twist of C-section

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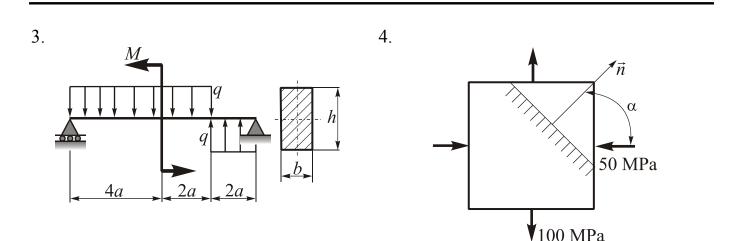
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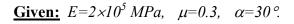
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 9

- 1. Tension-compression of prismatic rod. Deformations and displacements of crosssections. Condition of rigidity.
- 2. Principal axes of inertia. Extremity of geometrical properties of cross-section relative to principal axes.



- <u>Given:</u> a=2 m, $M=2qa^2$, $h=10 \times 10^{-2} mM$, h/b=2, $[\sigma]_t=200 MPa$, $[\sigma]_c=300 MPa$.
- Aim: determine [q].



<u>Aim</u>: calculate stresses on inclined α -plane and also relative change in volume ε_{ν} .

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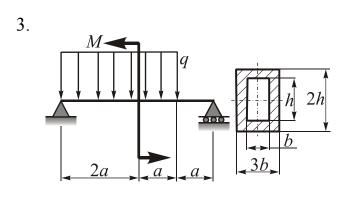
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Examination card № 10

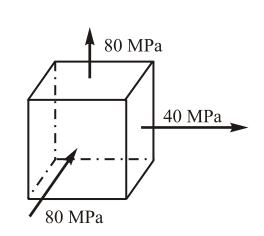
- 1. Statically indeterminate rod systems in tension-compression (an example of solving the problem of opening of static indeterminacy).
- 2. Method of sections and calculation of internal forces in cross-section, knowing external forces applied to elastic solid.

4.



<u>Given:</u> a=1 m, b=0.04 m, h=0.08 m, $M=2qa^2$, $[\sigma]_t=140 MPa$, $[\sigma]_c=200 MPa$.

<u>Aim</u>: determine [q].



<u>Given:</u> $\mu = 0,28$, $E = 2 \cdot 10^{11}$ Pa.

<u>Aim</u>: determine strains of the edges and relative change in volume ε_{v} .

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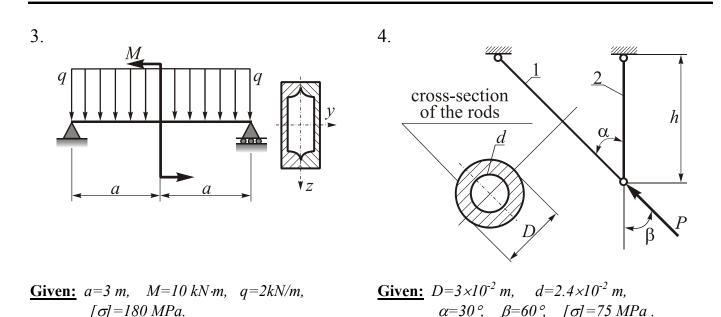
Examiner

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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 11

- 1. Thermal stresses in the rods and statically indeterminate rod systems in tensioncompression. Method of calculation (an example).
- 2. Allowable stress and its calculation in result of mechanical test.



<u>Aim:</u> determine № of channel section.

<u>Aim:</u> determine [P].

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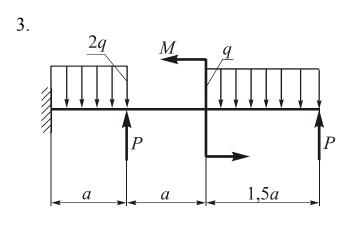
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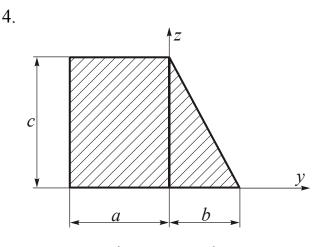
Examination card № 12

- 1. Assembly stresses in elements of statically indeterminate rod systems in tension-compression.
- 2. Maximum shearing stresses. Orientation of the planes of maximum shear stresses relative to principal planes and corresponding normal stresses.



<u>Given:</u> $a=2 \text{ } \text{ } \text{ } \text{ } P=4qa, M=qa^2, [\sigma]_c=200 \text{ } MPa,$ $[\sigma]_t=160 \text{ } MPa, I \text{ } N 20.$

<u>Aim</u>: determine allowable intensity of distributed load [*q*].



<u>Given:</u> $b=2\times 10^{-2} m$, $c=8\times 10^{-2} m$.

<u>Aim</u>: calculate value of *a* from the viewpoint that first moment of area relative to *z* axis equals to zero. Determine I_y , I_z , I_{yz} .

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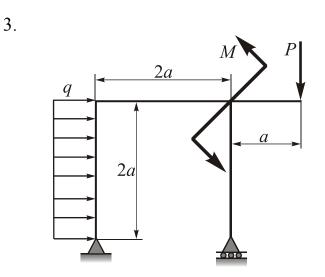
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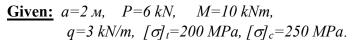
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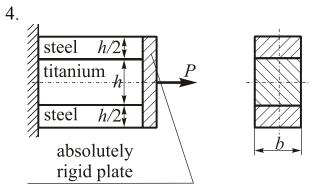
Examination card № 13

- 1. Torsional deformation. Proof of torsional formula (calculation of shear stresses in round solid and hollow shafts). General assumptions and hypotheses. Graph of shear stress distribution in cross-section, position of dangerous point, calculation of maximum stresses.
- 2. Method of sections. Determination of internal forces, knowing external ones (examples for three simple deformations).





<u>Aim:</u> draw graphs $N_x(x)$, $Q_z(x)$, $M_y(x)$; find number of I-beam section.



<u>**Given:**</u> Composite rod of rectangle cross-section consists of steel and titanium parts:

 $b=5\times10^{-2} m, \quad P=50 \ kN,$ $[\sigma]_{steel}=200 \ MPa, \quad [\sigma]_{titan}=600 \ MPa,$ $E_{steel}=2\times10^{5} \ MPa, \quad E_{titan}=1\times10^{5} \ MPa \ .$ Aim: find h.

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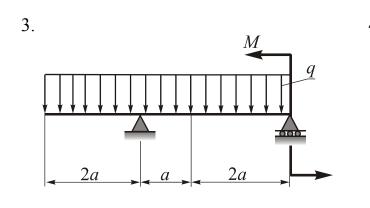
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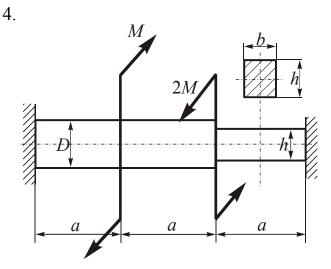
Examination card № 14

- 1. Torsional deformation. Proof of the formula for angle of twist of round solid and hollow shafts. Condition of rigidity.
- 2. Hooke's law in tension-compression. Hooke's law in pure shear. Relationship between *E* and *G* moduli.



<u>Given:</u> $a=2 m, M=2qa^2, [\sigma]_t=100 MPa, [\sigma]_c=200 MPa, I N \ge 10$.

<u>Aim</u>: determine allowable value of [q].



<u>Given:</u> $D=8\times10^{-2} m, b=3.0\times10^{-2} m, h=4.5\times10^{-2} m, M=10 kNm, [\tau]=80 MPa.$

Aim: check the strength of the shaft.

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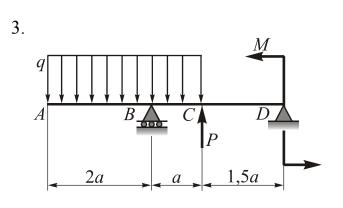
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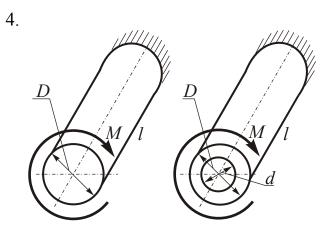
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 15

- 1. Torsion of the shaft with rectangle cross-section. Shear stress formula. Stress distribution in cross-section. Condition of strength. Particularities of shear stress calculation in thin-walled shafts of opened cross-section.
- 2. Force-elongation diagram of ductile and brittle materials. Stress-strain diagram. Mechanical properties of material determined in result of tension test.



- **<u>Given:</u>** $a=2 m, P=10 kN, M=3 kNm, [\sigma]_t=180 MPa, [\sigma]_c=200 MPa, q=2 kN/m.$
- **<u>Aim:</u>** determine $\mathbb{N}_{\mathbb{P}}$ of I-beam section and calculate τ_{max} .



<u>Aim</u>: estimate decrease of solid shaft allowable external torsional moment [M] relative to hollow one.

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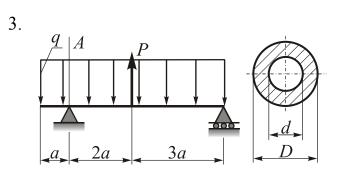
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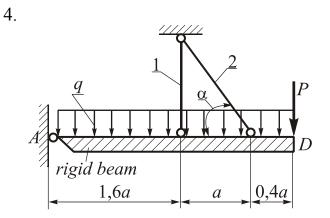
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 16

- 1. Torsion of shafts with rectangle cross-section. Formula for angle of twist and condition of rigidity. Particularities of angle of twist calculation for shafts of thin-walled opened cross-section.
- 2. Properties of material strength. Allowable stress as result of mechanical testing of structural material.



- <u>Given:</u> a=1 m, P=4 kN, q=2 kN/m, d/D=0.8, $[\sigma]_p=100 MPa$, $[\sigma]_c=200 MPa$.
- **<u>Aim:</u>** calculate *D* and draw the graph $\sigma_x(z)$ in *A*-section.



- **<u>Given:</u>** a=2 m, P=60 kN, q=10 kN/m, $[\sigma]=140 MPa$, $A_1=A_2=A$, $\alpha=60^\circ$, bar AD -absolutely rigid.
- <u>Aim:</u> 1. Determine normal forces in *1*, *2* rods;2. Calculate their cross-sectional area *A*.

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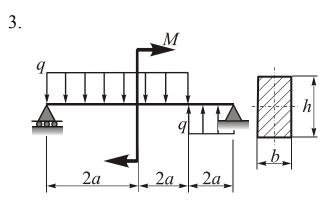
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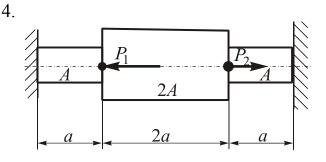
Examination card № 17

- 1. Torsion of thin-walled shafts with closed profile. Calculation of shear stresses. Condition of strength.
- 2. Generalized Hooke's law and its application for calculation of relative change in elastic material volume.



<u>Given:</u> a=1 m, $M=2qa^2$, $h=15 \times 10^{-2} m$, h/b=2, $[\sigma]=100 MPa$.

<u>Aim:</u> determine [q].



<u>Given:</u> $a=0.1 \text{ m}, P_1=20 \text{ kN}, P_2=10 \text{ kN}, [\sigma]_c=160 \text{ MPa}, [\sigma]_t=100 \text{ MPa}.$

<u>Aim:</u> draw graphs $N_x(x) \sigma_x(x)$; determine cross-sectional area *A*.

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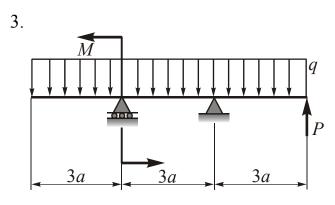
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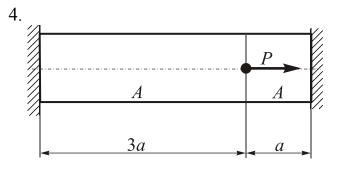
Examination card № 18

- 1. Torsion of the shafts with round solid cross-section. Proof of angle of twist formula. Condition of rigidity.
- 2. Explain the terms: "combined stress-state", "stress tensor", "strain tensor".



<u>Given:</u> a=1 m, $M=4qa^2$, P=5qa, $[\sigma]=200 MPa$, I $N \ge 18$.

<u>Aim</u>: determine [q].



<u>Given:</u> $[\sigma] = 200 \text{ MPa}, A = 10 \times 10^{-4} \text{ m}^2, a = 2 \text{ m}, E = 2 \times 10^5 \text{ MPa}, \alpha_t = 1.25 \times 10^{-5} \text{ 1/K}.$

<u>Aim:</u> determine change of [*P*] after 50 K change of rod temperature.

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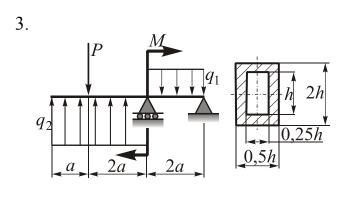
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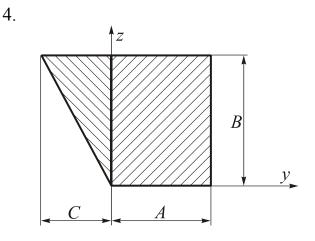
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Examination card № 19

- 1. Opening of static indeterminacy of shaft in torsion (describe the example).
- 2. Define the concepts of principal plane and principal stress. Types of stress state in vicinity of a point.





<u>Given:</u> a=1 m, M=4 kNm, P=6 kN, $q_1=2 kN/m$, $q_2=1 kN/m$, $\lceil \sigma \rceil = 100 MPa$.

<u>Aim</u>: determine *h*.

<u>Given:</u> $A=10\times 10^{-2}$ m, $B=6\times 10^{-2}$ m.

<u>Aim</u>: calculate value of *C* from the viewpoint that first moment of area relative to z axis equals to zero. Determine I_y , I_z , I_{yz} .

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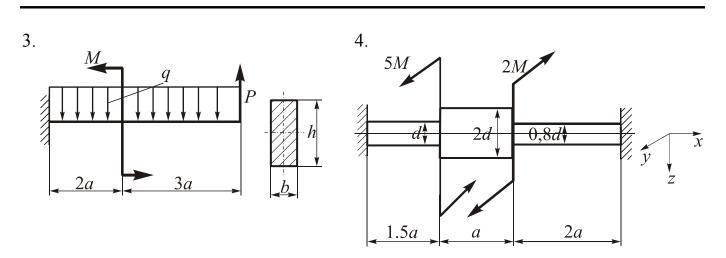
Examiner

Fomichov P.O.

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 20

- 1. Types of bending. Bending formula for prismatic beam in pure plane bending (proof). Condition of strength.
- 2. Principal hypotheses and assumptions in mechanics of materials.



<u>Given:</u> a=1 m, h=1.25b, P=4 kN, M=6 kNm, q=2 kN/m, $[\sigma]=120 MPa$.

<u>Aim</u>: determine *b* and *h* dimensions of beam cross-section.

Aim: determine *d*.

 $M=10 \ kNm$

<u>Given:</u> $a=0.1 m, d=4 \times 10^{-2} m, [\tau]=80 MPa,$

 $[\phi]=0.5 \text{ degree/m}, G=8\times10^4 \text{ MPa}.$

Accepted by Department of Aircraft Strength meeting.

Record of proceeding N_{23} , 21 November, 2011

Head of Department, Doctor of Science, Professor

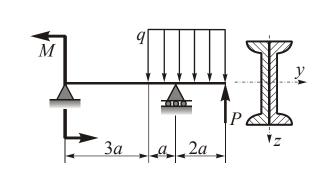
Examiner

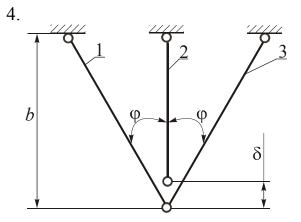
Fomichov P.O.

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 21

- 1. Transverse plane bending of prismatic beam. Proof of Juravsky formula for shear stress distribution over the cross-section. Proof of possibility of pure bending formula use for studying the normal stress distribution in transverse bending.
- 2. Concept of strain energy in elastic deformation.





- <u>Given:</u> a=1 m, q=10 kN/m, M=6 kNM, P=10 kN, $[\sigma]=180 MPa$.
- <u>Aim:</u> determine the number of channel section for composite beam.

<u>Given:</u> $b=10\times10^{-2} \text{ m}, \quad \delta=0.2\times10^{-3} \text{ m},$ $A_1=A_2=A_3=2\times10^{-4} \text{ m}^2,$ $E_1=E_2=E_3=2\times10^5 \text{ MPa}, \ \varphi=30^\circ.$

Aim: calculate stresses in rods after assembly.

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Head of Department, Doctor of Science, Professor

Examiner

3.

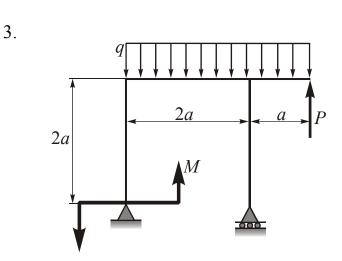
Fomichov P.O.

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

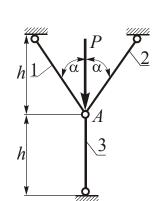
Examination card № 22

- 1. Differential relationships between q(x), $Q_z(x)$, $M_y(x)$ in plane bending (proof) and their application for checking the graphs of internal forces distribution (an example).
- 2. Hypothesis of plane sections and its use in proof of general formulae of tensioncompression, torsion, bending.

4.



- <u>Given:</u> a=0.5 m, M=6 kNm, q=18 kN/m, P=10 kN, $\lceil \sigma \rceil = 200 MPa$.
- <u>Aim:</u> draw graphs $N_x(x)$, $Q_z(x)$, $M_y(x)$; determine the diameter of round cross-section.



<u>Given:</u> $h=1 \text{ } M, \quad A_1=A_2=A, \quad A_3=2A,$ $\alpha=30^\circ, \quad [\sigma]=140 \text{ } MPa, \quad P=40 \text{ } kN,$ $E_1=E_2=E_3=E=2\times 10^5 \text{ } MPa$.

<u>Aim:</u> determine cross-sectional area *A* and vertical displacement of the hinge *A*.

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Head of Department, Doctor of Science, Professor

Examiner

Fomichov P.O.

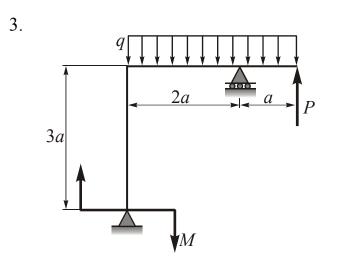
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 23

1. Proof of differential equation of beam deflected curve. Determination of constants of integration.

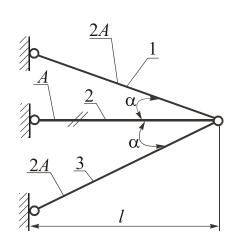
4.

2. Poisson's ratio and the method of its experimental determination.



<u>Given:</u> a=1 m, M=6 kNm, q=10 kN/m, P=10 kN, $[\sigma]=160 MPa$.

<u>Aim:</u> draw graphs $N_x(x)$, $Q_z(x)$, $M_y(x)$; determine the diameter of the frame.



<u>Given:</u> rod 2 is heated at 60 K, $\alpha_t = 1.25 \times 10^{-5} \ 1/K, \quad E = 2 \times 10^5 \ MPa,$ $A = 2 \times 10^{-4} \ m^2, \quad l = 2 \ m, \quad \alpha = 30^{\circ}.$ <u>Aim:</u> determine stresses in the rods 1, 2, 3.

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Record of proceeding N_{23} , 21 November, 2011

Head of Department, Doctor of Science, Professor

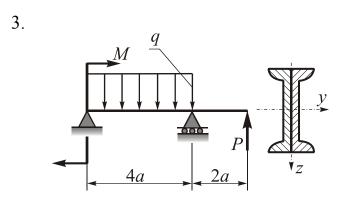
Fomichov P.O.

Examiner

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

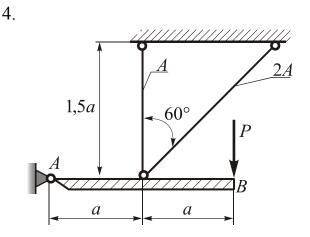
Examination card № 24

- 1. Generalized Hooke's law. Proof of its equations in the terms of principal stresses.
- 2. Geometrical and physical sense of modulus of elasticity in tension-compression. Method of its experimental determination.



<u>Given:</u> $a=1 \text{ } m, \quad q=10 \text{ } kN/m, \quad M=12 \text{ } kNm, \quad P=10 \text{ } kN, \quad [\sigma]=180 \text{ } MPa.$

<u>Aim:</u> draw graphs $Q_z(x)$, $M_y(x)$ and determine \mathbb{N}_{2} of channel in composite section. Calculate τ_{max} in critical section.



<u>Given:</u> $a=1 m, A=10 \times 10^{-4} m^2, [\sigma] = 200$ MPa.

Aim: calculate allowable value of *P* force.

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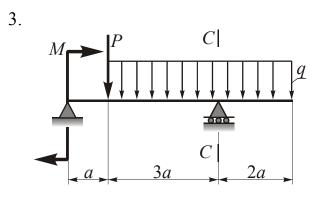
Examiner

Fomichov P.O.

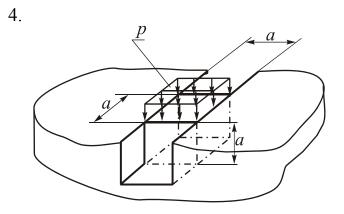
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 25

- 1. Bending formula in pure bending (proof). Normal stress distribution, critical points of cross section, condition of strength.
- 2. Geometrical and physical sense of shear modulus of elasticity. Method of its experimental determination.



- **<u>Given:</u>** a=1 m, q=2 kN/m, P=12 kN, M=6 kNm, $[\sigma]=200 MPa$.
- <u>Aim:</u> determine number of I-beam section and τ_{max} in C section



- **<u>Given:</u>** $a=10 \times 10^{-3} m$, P=100 MPa, $E=1 \times 10^{5} MPa$, $\mu=0.32$.
- <u>Aim:</u> determine stresses on the faces of the element and also strains of its edges.

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Head of Department, Doctor of Science, Professor

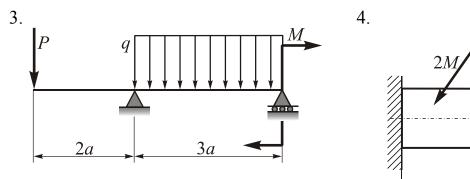
Fomichov P.O.

Examiner

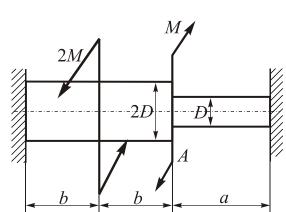
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 26

- 1. Shear stresses in transverse bending. Proof of Juravsky formula.
- 2. Tension tests of materials and calculation of allowable stresses.



- **<u>Given:</u>** a=0.5 m, P=6 kN, q=2 kN/m, M=4 kNm, $[\sigma]=200 MPa$.
- <u>Aim:</u> determine diameter of round cross-section and draw the graph $\tau(z)$ in *A*-section.



<u>Given:</u> a=1.8 m, b=1.2 m, M=30 kNm, $G=8\times10^4 MPa, [\psi]=0.5 degree/m.$

<u>Aim</u>: determine D and angle of twist of A-section.

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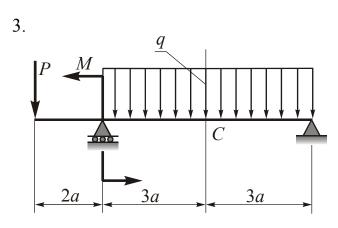
Fomichov P.O.

Examiner

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

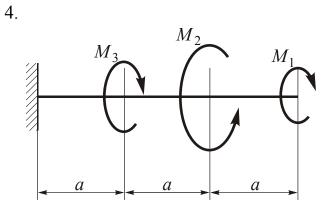
Examination card № 27

- 1. Stresses at an arbitrary point of cross-section in plane transverse bending. Proof of the bending formula and estimation of shear stress distribution in round solid and hollow cross-sections.
- 2. First moment of area. Central axes and coordinates of geometrical center (centroid) of cross-section.



<u>Given:</u> a=0.5 m, P=20 kN, M=40 kNm, q=20 kN/m, $[\sigma]=100 MPa$.

<u>Aim:</u> determine number of I-beam section and draw graph $\sigma_x(z)$ in *C*-section. Calculate τ_{max} .



<u>Given:</u> a=0.4 m, $M_1=20 kNm$, $M_2=80 kNm$, $M_3=10 kNm$, $[\tau]=100 MPa$, $G=8\times10^4 MPa$, $[\psi]=0.5 degree/m$.

<u>Aim:</u> determine diameter of the shaft and draw the graph of angle of twist distribution.

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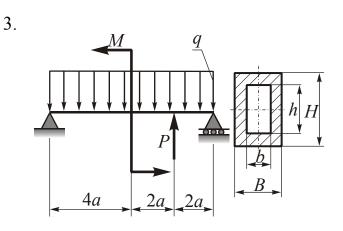
Examiner

Fomichov P.O.

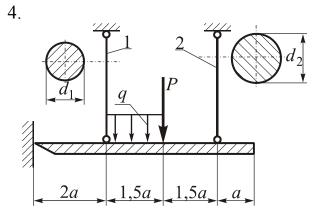
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 28

- 1. Formula of shear stresses in torsion of solid shaft (proof). Condition of strength in torsion.
- 2. Methods of experimental study of strains.



<u>Given:</u> a=1 m, $M=2qa^2$, P=4qa, $H=8\times10^{-2} m$, $h=6\times10^{-2} m$, $B=6\times10^{-2} m$, $b=4\times10^{-2} m$, $[\sigma]=100 MPa$.



<u>Given:</u> $a=1 m, P=100 kN, d_1/d_2=0.8, q=20 kN/m, [\sigma]=100 MPa$.

<u>Aim</u>: determine d_1, d_2 .

<u>Aim</u>: determine [q].

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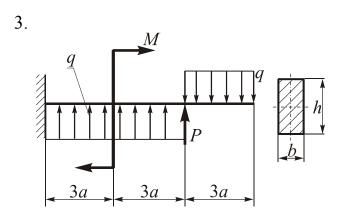
Fomichov P.O.

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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

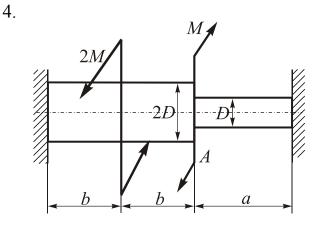
Examination card № 29

- 1. Methods of experimental study of mechanical properties of structural materials. Properties of material strength and ductility (plasticity).
- 2. Features of strength analysis of statically indeterminate rod systems in tension-compression.



<u>Given:</u> a=0.25 m, M=10 kNm, q=2 kN/m, P=10 kN, $b=5 \times 10^{-2} m$, h/b=2, $\lceil \sigma \rceil = 100 MPa$.

<u>Aim</u>: check the strength of the beam and draw graphs $\sigma_x(z)$ and $\tau_{xz}(z)$ in rigid support.



<u>Given:</u> a=2 m, b=1.5 m, M=30 kNm, $[\tau]=80 MPa, G=8 \times 10^4 MPa,$

Aim: calculate *D* and angle of twist of *A*-section.

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Head of Department, Doctor of Science, Professor

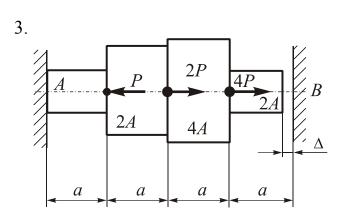
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Fomichov P.O.

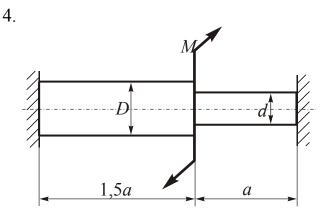
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 30

- 1. Proof of the formula for shear stress distribution over the cross-section of a beam in plane transverse bending..
- 2. Show the difference between "force-elongation" and "stress-strain" diagrams.



<u>Given:</u> $a=0.5 m, A=10 cm^2$ $[\sigma]=100 MPa, P=20kN, \Delta=0.2 mm.$



<u>Given:</u> $a=0.6 m, D=4 \times 10^{-2} m, d/D=0.8,$ [τ]=90 MPa.

<u>Aim:</u> check the strength.

Aim: check the strength of the shaft.

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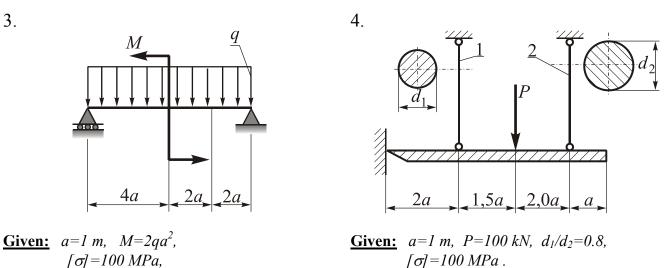
Fomichov P.O.

Examiner

National aerospace university "Kharkiv Aviation Institute" Branch of education: 1001 Degree B. Sc. **Aerospace Engineering** Semester III Course "Mechanics of materials"

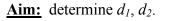
Examination card № 31

- Method of stress analysis of statically indeterminate shafts in torsion (an example). 1.
- Allowable stress and its determination on the basis of tensile test. 2.



 $[\sigma] = 100 MPa$, cross-section – I-beam № 20.

<u>Aim</u>: calculate [q].



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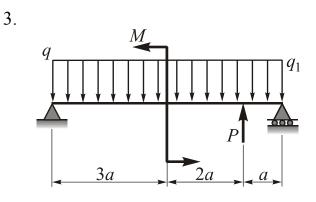
Fomichov P.O.

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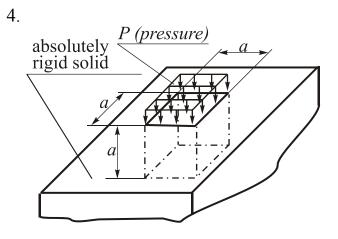
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 32

- 1. Proof of the formula for acting stresses in pure plane bending and also method of strength analysis of the beam in pure plane bending.
- 2. Poisson's ratio and method of its experimental determination.



- <u>Given:</u> a=1 m, M=40 kNM, P=10kN, $[\sigma]=200 MPa$, q=20 kN/m.
- <u>Aim:</u> determine № of I-beam section in plane bending.



- **<u>Given:</u>** $a=0.1 \times 10^{-2} m$, P=100 MPa, $\mu=0.28$, $E=2 \times 10^{5} MPa$.
- <u>Aim:</u> calculate relative change in volume of elastic element put into absolutely rigid plate without gaps.

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Record of proceeding № 3, 21 November, 2011

Head of Department, Doctor of Science, Professor

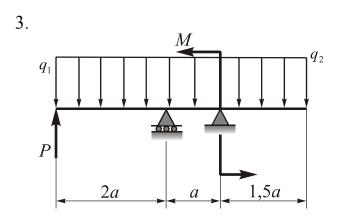
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Fomichov P.O.

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

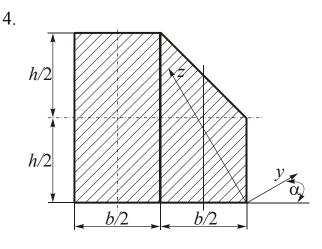
Examination card № 33

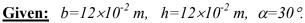
- 1. Experimental investigation of mechanical properties of structural materials. Factors of material ductility and strength.
- 2. Torsion of rectangle shafts. Critical points and stresses, condition of strength.



Given: a=1 m, M=20 kNm, P=20kN, q=30kN/m $[\sigma]=200 MPa$,

<u>Aim</u>: determine number of I-beam section and calculate τ_{max} .





<u>Aim:</u> calculate axial moments of inertia and also product of inertia of the section relative to *y*, *z* axes.

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Head of Department, Doctor of Science, Professor

Fomichov P.O.

Demenko V.F.

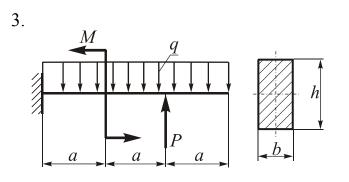
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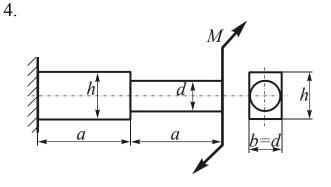
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 34

- 1. Parallel transfer of axes of inertia (proof). Calculation of axial moments of inertia relative to axes which are parallel to original central ones.
- 2. Factors (characteristics) of material strength. Method of determination of allowable stresses.



- <u>Given:</u> a=3 m, q=2 kN/m, P=10kN, M=6 kNm, $[\sigma]=100 MPa$, h/b=2.
- <u>Aim:</u> determine dimensions of rectangle crosssection and also draw graphs σ_x and τ_{xz} distribution in critical section.



- **<u>Given:</u>** h=2b, $b=d=4 \times 10^{-2}$ m, $[\tau] = 80$ MPa.
- <u>Aim:</u> determine allowable external torsional moment [M].

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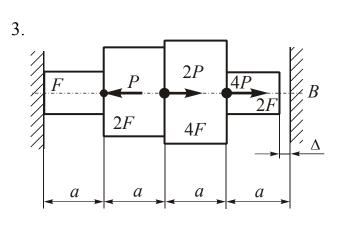
Fomichov P.O.

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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

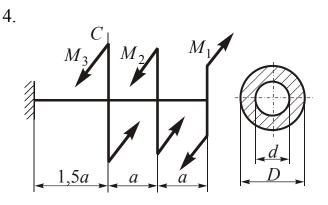
Examination card № 35

- 1. Rotation of axes of inertia. Proof of the formula of product of inertia in rotation of axes. Determination of principal axes position and calculation of principal moments of inertia.
- 2. Relationships between internal forces and internal stresses in cross-section for simple deformations.



<u>Given:</u> a=1 m, $\Delta=0.3 mm$, $A=10cm^2$, P=60kN.

<u>Aim:</u> determine stresses in cross-sections of the rod



<u>Given:</u> a=0.5 m, $\alpha=d/D=0.8$, $G=8\times10^4 MPa$, $[\theta]=1 \ degree/m$, $M_1=40 \ kNm$, $M_2=8 \ kNm$, $M_3=10 \ kNm$.

<u>Aim:</u> calculate *d* and *D* of the shaft and also angle of twist of *C*-section.

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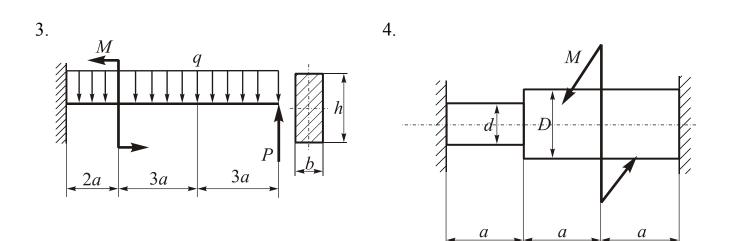
Fomichov P.O.

Examiner

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 36

- 1. Plane stress state at a point of deformable solid. Proof of the formulae for calculation of stresses in the plane of general position in plane stress state.
- 2. Properties of material ductility in tension test.



<u>Given:</u> a=1 m, q=2 kN/m, P=6 kN, $M=4 kNm, [\sigma]=200 MPa.$

<u>Aim</u>: determine h.

<u>Given:</u> $D=8\times 10^{-2}$ m, $d=4\times 10^{-2}$ m, a=1 m, $G=0.8\times 10^{11}$ Pa, M=10 kNm, $[\tau]=80$ MPa.

Aim: check the strength of the shaft.

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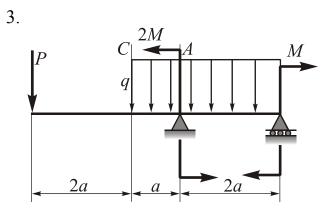
Fomichov P.O.

Examiner

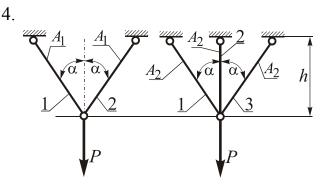
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 37

- 1. Stresses at the plane of general position in plane stress state. Proof of the formula for principal stresses and positions of principal planes.
- 2. Principle of superposition.



- <u>Given:</u> a=1 m, q=2 kN/m, P=12 kN, M=6 kNm, $[\sigma]=200 MPa$. h/b=2
- <u>Aim:</u> determine dimensions of rectangle cross section and calculate τ_{max} in *C*-section.



- <u>Given:</u> h=1 m, P=20kN, $\alpha=30^{\circ}$, $[\sigma]=160 MPa$, $\rho=7.8 \times 10^{3} kg/m^{3}$.
- **<u>Aim:</u>** determine areas A_1 , A_2 for two rod systems and compare their masses.

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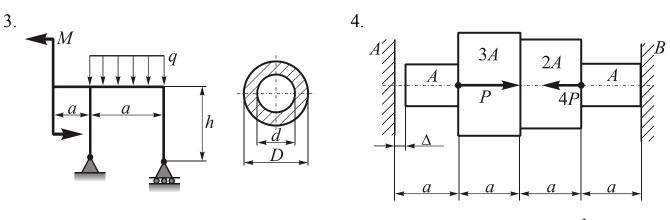
Fomichov P.O.

Examiner

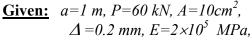
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 38

- 1. Mohr's circle in solution of direct and inverse problems of plane stress state (essence of the method and examples of solution).
- 2. External and internal forces, their types and units.



- <u>Given:</u> a=1 m, h=2 m, q=2 kN/m,M=8 kNm, $[\sigma]=100 MPa, d/D=0.8.$
- **<u>Aim:</u>** draw graphs N_x , Q_z , M_y ; calculate D.



<u>Aim</u>: draw graphs $N_x(x)$ and $\sigma_x(x)$.

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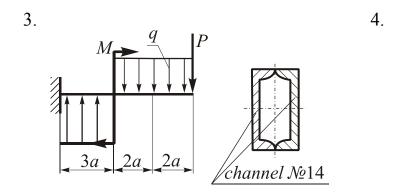
Fomichov P.O.

Examiner

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

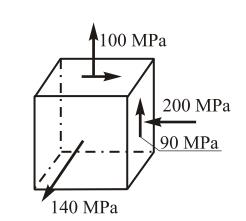
Examination card № 39

- 1. Proof of generalized Hooke's law equations.
- 2. Principles of rational orientation of specified cross-section in plane bending.



<u>Given:</u> a=2 m, P=2qa, $M=4qa^2$, $\lceil \sigma \rceil = 200 MPa$.

<u>Aim</u>: determine [q].



<u>Given:</u> $E=2 \times 10^5 MPa$, $\mu=0.3$.

<u>Aim</u>: determine position of principal planes and principal stresses and also relative change in volume ε_{ν} .

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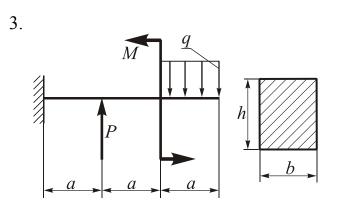
Examiner

Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 40

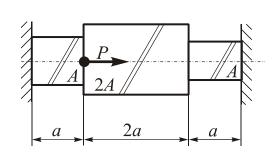
- 1. Strain state at a point of elastic deformable solid. Relationships between displacements and components of strain state (Cauchy equations).
- 2. Sectional moduli of cross-sections (definition and examples of determination for simple cross-sections).

4



<u>Given:</u> a=1 m, M=30 kNm, q=40 kN/m, P=20 kN, $b=5 \times 10^{-2} m$, $h=10 \times 10^{-2} m$, $\lceil \sigma \rceil = 100 MPa$.

<u>Aim</u>: calculate σ_{max} and check the strength of the beam.



<u>Given:</u> $a=1 m, P=80 kN, [\sigma]_t=160 MPa, [\sigma]_c=100 MPa, A=10 cm^{2}, E=2 \times 10^{11} Pa, \alpha_t = 17.5 \times 10^{-6} 1/K, \Delta t = 50 \%$

Aim: check the strength of the rod.

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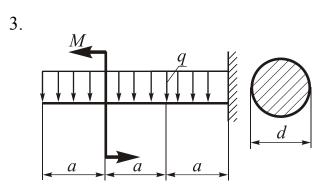
Examiner

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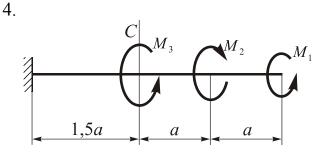
Examination card № 41

- 1. Tension-compression of prismatic rod. Hypothesis of plane sections. Normal stresses and condition of strength.
- 2. Axial and polar moments of inertia and also product of inertia: definitions and relationships between axial and polar moments of inertia.



<u>Given:</u> $a=1 m, d=8 \times 10^{-2} m,$ [σ]=180 MPa, M=4qa²

<u>Aim:</u> draw graphs Q_z , M_y and determine allowable value of intensity of external load [q].



<u>Given:</u> $G=8 \times 10^4 MPa$, $[\psi]=1 degree/m$, a=1.0 m, d/D=0.8, $M_1=20 kNm$, $M_2=15 kNm$, $M_3=8 kNm$,

<u>Aim</u>: draw graph $M_x(x)$, determine diameters of hollow shaft and also angle of twist in *C*-section.

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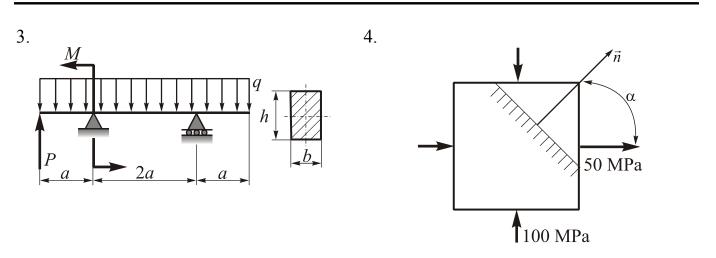
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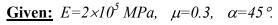
Examination card № 42

- 1. Tension-compression of prismatic rod. Calculation of strains and displacements. Condition of rigidity.
- 2. Principal axes of inertia. Properties (extremity) of geometrical characteristics of the cross-section relative to principal axes.



<u>Given:</u> a=2 m, M=20 kNm, P=30 kN, h/b=2, $[\sigma]=200 MPa$.

Aim: determine h, b.



<u>Aim</u>: determine acting stresses at inclined plane and also relative change in volume ε_{v} .

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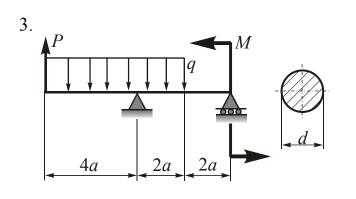
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Examination card № 43

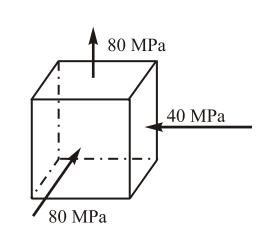
1. Statically indeterminate rod systems in tension-compression (examples of the opening of statical indeterminacy).

4.

2. Method of sections as the principle of determination of internal forces.



<u>Given:</u> a=1 m, M=60kNm, P=80 kN, q=10 kN/m $[\sigma]=140 MPa$.



Given: $\mu = 0,28$, $E = 2 \cdot 10^{11} Pa$.

<u>Aim:</u> determine deformations of element's edges and relative change of its volume.

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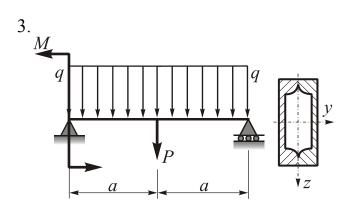
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Aim: determine *d*.

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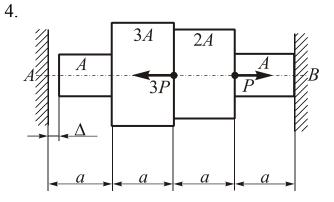
Examination card № 44

- 1. Thermal stresses in statically indeterminate rod systems in tension-compression (illustration of the method of internal forces calculation in example).
- 2. Allowable stress and its determination for ductile and brittle materials.



<u>Given:</u> a=3 m, M=20 kNm, q=40 kN/m, P=10kN, $[\sigma]_c=200 MPa$, $[\sigma]_t=160 MPa$

<u>Aim:</u> determine number of the channel in box composite section



<u>Given:</u> $a=1 m, \Delta=0.1 \times 10^{-2} m, A=10 \times 10^{-4} m^2,$ $P=40 kN, [\sigma]=200 MPa$.

Aim: determine stresses and check the strength.

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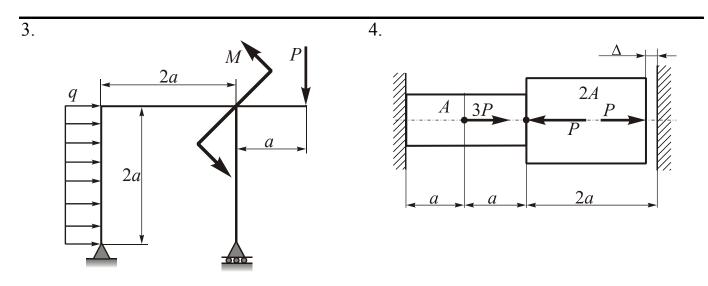
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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 45

- 1. Torsional deformation. Proof of the torsional formula for stresses in round and hollow shafts. General assumptions. Geometrical presentation of stress distribution.
- 2. Essence of method of sections and general goal of its application in mechanics of materials.



<u>Given:</u> a=2 m, P=6 kN, M=10 kNm, q=10 kN/m

q 10 m m

 $P=20 \text{ kN}, A=20 \times 10^{-4} \text{ m}^2,$ [σ]=200 MPa, E=2×10⁵ MPa.

Given: $\Delta = 0.1 \text{ mm}, a = 1 \text{ m},$

<u>Aim:</u> determine acting stresses and check the strength.

<u>Aim</u>: draw graphs N_x , Q_z , M_y ;

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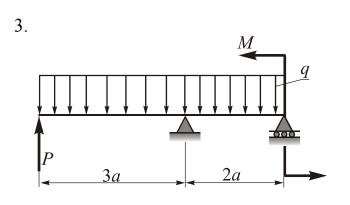
Fomichov P.O.

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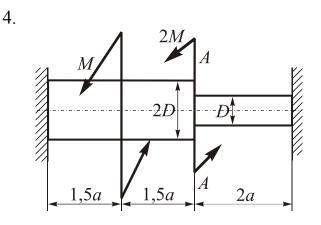
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 46

- 1. Torsional deformation. Formula for the angle of twist for round and hollow shafts (proof). Condition of rigidity.
- 2. Hooke's law in tension-compression and pure shear.



- **<u>Given:</u>** a=2 m, $M=2qa^2$, $[\sigma]=100 MPa$, I $\mathcal{N}_2 10$.
- <u>Aim:</u> calculate allowable value of external loading [q].



<u>Given:</u> $D=8 \times 10^{-2} m$, a=1 m, $G=0.8 \times 10^{11} Pa$ M=10 kNm, $[\tau]=80 MPa$.

Aim: check the strength of the shaft.

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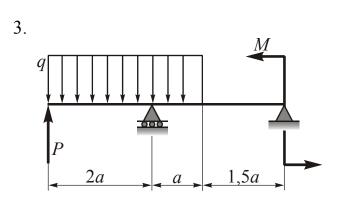
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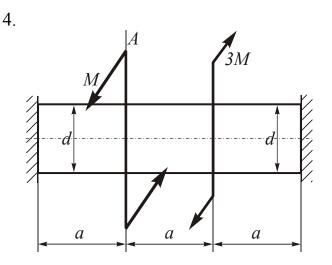
Examination card № 47

- 1. Torsion of the shaft with rectangle cross-section. Formula for maximum shear stresses. Position of critical points and condition of strength.
- 2. "Load-displacement" and "stress-strain" diagrams in tension of ductile material. Mechanical properties, which are calculated from "stress-strain" diagram. Conventional yield limit.



<u>Given:</u> a=2 m, M=30 kNm, P=10 kN, q=2 kN/m, $[\sigma]=180 MPa$,

<u>Aim:</u> determine the number of I-beam section and calculate τ_{max} .



<u>Given:</u> a=2 m, d=8 cm, $[\tau]=80 MPa$

<u>Aim:</u> calculate allowable value of external moment [M].

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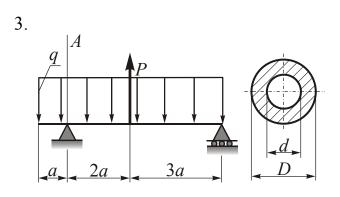
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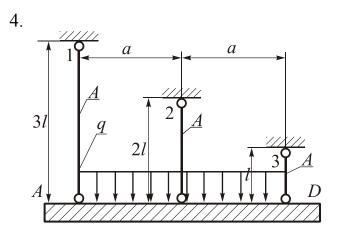
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 48

- 1. Torsional deformation of the shafts with rectangle cross-section. Formula for angle of twist and condition of rigidity.
- 2. Characteristics of material strength. Method of calculation of allowable stresses.



- <u>Given:</u> a=1 m, P=40 kN, q=20 kN/m, d/D=0.8, $[\sigma]=100 MPa$.
- <u>Aim</u>: calcilate *D*; draw the graph $\sigma_x(z)$ in critical cross-section.



- **<u>Given:</u>** a=2 m, q=10 kN/m, $E=2 \times 10^5 MPa, [\sigma]=140 MPa,$ $A_1=A_2=A_3=A,$ AD – absolutely rigid beam.
- Aim:1. Determine internal forces in the rods1, 2, 3;2. Determine cross-sectional area A.

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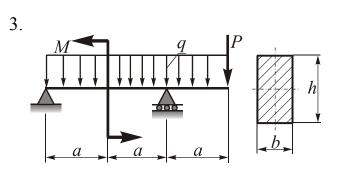
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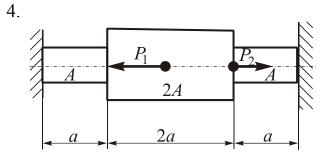
Examination card № 49

- 1. Torsional deformation of solid round shafts. Torsional formula (proof). Condition of strength.
- 2. Generalized Hooke's law and its application for calculation of the relative change of elastic material volume.



<u>Given:</u> a=1 m, M=30 kNM, q=20 kN/m, P=20 kN, h=10 cm, b=5 cm, $[\sigma]=200 MPa$.

<u>Aim:</u> check the strength, calculate τ_{\max} .



<u>Given:</u> a=1 m, $P_1=30 kN$, $P_2=10 kN$, $[\sigma]=160 MPa$, $A=20cm^2$. <u>Aim:</u> determine internal force N_x , calculate acting stresses σ_x , *F*. check the strength of a rod.

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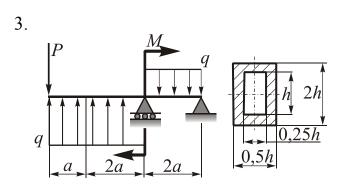
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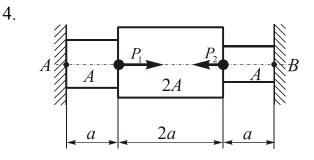
Examination card № 50

- 1. Method of opening of statical indeterminacy of shafts (describe an example).
- 2. Concepts of principal planes stresses and principal strains. Types of stress state at a point of deformable solid.



<u>Given:</u> a=1 m, M=40 kNm, P=10 kN, q=20 kN/m, $[\sigma]=200 MPa$.

<u>Aim:</u> calculate *h*.



<u>Given:</u> $A=10\times 10^{-4} m^2$, $a=1 m, P_1=20 kN$, $P_2=50 kN, E=2\times 10^{11} Pa$.

Aim: calculate stresses in the rod.

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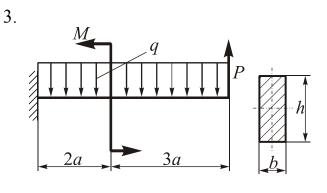
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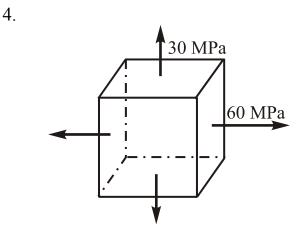
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Examination card № 51

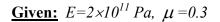
- 1. Types of bending. Bending formula for normal stresses in cross-section of prismatic bar in pure bending (proof). Condition of strength, types of engineering problems which are solved using condition of strength.
- 2. General hypotheses in mechanics of materials.





<u>Given:</u> a=1 m, h=2b, P=40 kN, M=30 kNm, q=20 kN/m, $\lceil \sigma \rceil = 160 MPa$.

Aim: calculate *b*, *h*



<u>Aim:</u> calculate strains of the element edges and relative change in volume.

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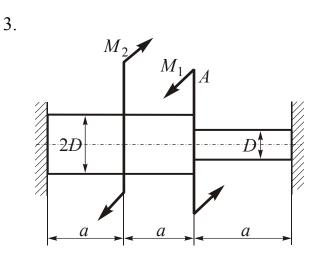
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 52

1. Transverse bending of prismatic bar. Proof of Juravsky formula for shear stresses in transverse bending. The graphs of shear stress distribution in round, rectangle and I-beam sections.

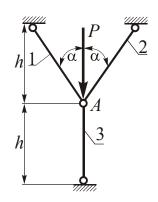
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2. Elastic moduli and their correlation.



<u>Given:</u> a=1 m, $M_1=10 kNm$, $M_2=40 kNm$, $[\tau]=80 MPa$, $G=0.8 \times 10^{11} Pa$.

Aim: calculate diameter D.



Given:
$$b=100 \text{ cm}, \quad \delta=0.02 \times 10^{-2} \text{ m},$$

 $A_1=A_2=A_3=2 \times 10^{-4} \text{ m}^2,$
 $E_1=E_2=E_3=2 \times 10^5 \text{ MPa}, \quad \varphi=30^{\circ}.$

<u>Aim:</u> calculate stresses in the rods after assembly.

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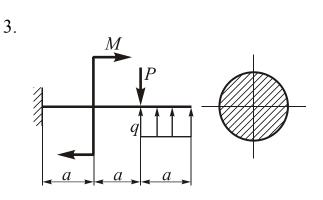
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 53

1. Differential relationships between q(x), $Q_z(x)$, $M_y(x)$ in plane bending (proof) and their application for checking the accuracy of internal force diagrams.

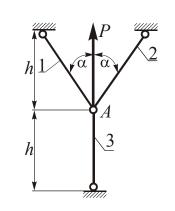
4.

2. Hypothesis of plane sections and its application in proof of bending formula.



<u>Given:</u> a=1 m, M=60 kNm, q=20 kN/m, p=10 kN, $[\sigma]=200 MPa$.

Aim: calculate the diameter of the beam.



<u>Given:</u> h=1 m, $A_1=A_2=A$, $A_3=2A$, $\alpha=30^{\circ}$, $[\sigma]=160 MPa$, P=40 kN, $E_1=E_2=E_3=E=2\times10^5 MPa$.

Aim: calculate cross-sectional area A.

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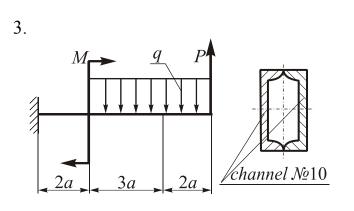
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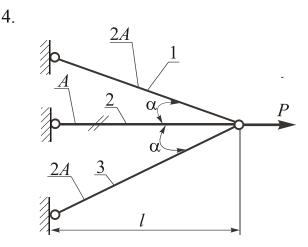
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 54

- 1. Proof of the differential equation of deflection curve. Calculation of the constants of integration.
- 2. Poisson's ratio and the method of its experimental determination.



- **<u>Given:</u>** a=1 m, M=20 kNm, q=10 kN/m, P=10kN, composite cross-section consisting of two channels $N \ge 10$, $[\sigma] = 160 MPa$.
- <u>Aim:</u> check the strength of the cantilever beam and find the largest shear stresses.



<u>Given:</u> central rod 2 is heated at 60 K, *P*=10 kN, $\alpha_t = 1.25 \times 10^{-5} \ 1/K, \ E = 2 \times 10^{5} \ MPa,$ $A = 5 \times 10^{-4} \ m^2, \ l = 2 \ m, \ \alpha = 30^{\circ},$ $\lceil \sigma \rceil = 160 \ MPa.$

<u>Aim:</u> check the strength of the rods.

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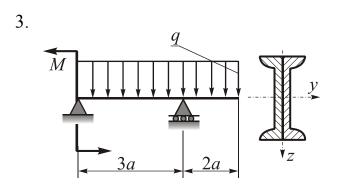
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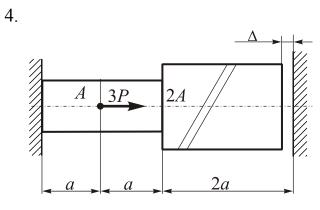
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 55

- 1. Proof of generalized Hooke's law equations.
- 2. Geometrical and physical sense of elasticity modulus. Method of its experimental study.



<u>Given:</u> a=1 m, q=10 kN/m, M=12 kNm, $[\sigma]=180 MPa$.



<u>Given:</u> $a=1 m, A=10 \times 10^{-4} m^2, [\sigma] = 200 MPa$ $\alpha = 1.75 \times 10^{-5} 1/K, E=2 \times 10^{11} Pa,$ $\Delta = 0.2 mm, P=40 kN, \Delta t = 50 °K.$

<u>Aim:</u> draw the graphs of $Q_z(x)$, $M_y(x)$; determine the number of channel section as the half of composite section; calculate σ_{max} and τ_{max} .

Aim: check the strength of the rod.

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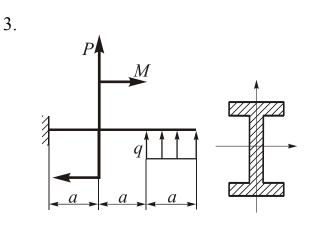
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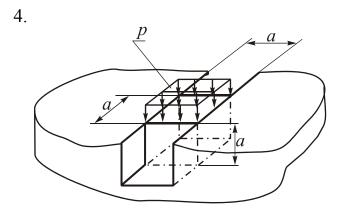
Examination card № 56

- 1. Torsional deformation of solid round and hollow shafts. Formula for angle of twist (proof). Condition of rigidity.
- 2. Generalized Hooke's law and its application for calculation of stresses, strains and relative change of elastic material volume (unit volume change).



<u>Given:</u> a=1 m, q=20 kN/m, P=10 kN, M=60 kNm, $[\sigma]=200 MPa$.

Aim: determine the number of I-beam section.



<u>Given:</u> $a=10 \times 10^{-3}$ m, P=60 MPa, $E=2 \times 10^{5}$ MPa, $\mu=0.30$.

<u>Aim:</u> calculate: a) stresses on the faces of the element; b) strains of its ages; c) relative change of element volume.

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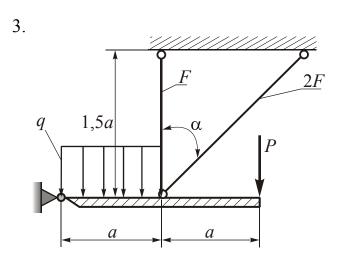
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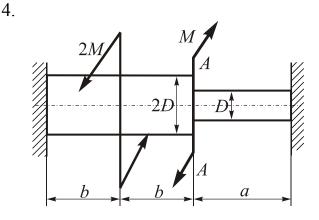
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 57

- 1. Shear stresses in transverse bending. Proof of the Juravsky formula. Graphs of shear stress distribution for round solid, round hollow and rectangle cross-section.
- 2. Tension test of material, mechanical properties as the results of the test, allowable stresses.



- **<u>Given:</u>** a=1 m, P=10 kN, q=20 kN/m, $[\sigma]=200 MPa$, $A=10 cm^2$, $\alpha = 30 %$
- Aim: calculate cross-sectional area A.



<u>Given:</u> a=2 m, b=1.2 m, M=30 kNm, $G=8 \times 10^4 MPa, [\psi]=0.5 degree/m.$

<u>Aim</u>: calculate D and angle of twist of A cross-section.

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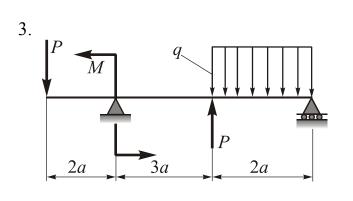
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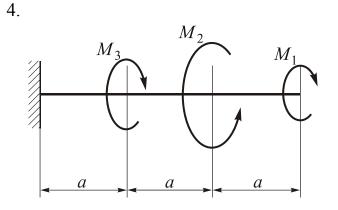
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 58

- 1. Formula for calculation of normal stresses at an arbitrary point of the beam in transverse bending (proof). Maximum bending stresses, condition of strength.
- 2. First moment of area. Central axes. Method of calculation of cross-sectional centroid (an example).



- <u>Given:</u> a=0.5 m, P=20 kN, M=40 kNm, q=20 kN/m, $[\sigma]=200 MPa$.
- **<u>Aim</u>**: determine the dimensions of I-beam section (number) and draw the graph of σ_x distribution in *C*-section.



<u>Given:</u> a=1 m, $M_1=20 kNm$, $M_2=50 kNm$, $M_3=10 kNm$, $[\tau]=100 MPa$, $G=8\times10^4 MPa$, $[\psi]=0.5 degree/m$.

Aim: calculate the diameter of solid shaft.

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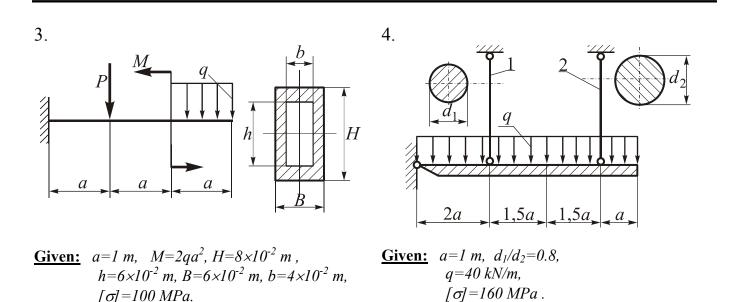
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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 59

1. Differential relationships between q(x), $Q_z(x)$, $M_y(x)$ in plane bending (proof) and their application for checking the graphs of internal forces distribution (an example).

2. Methods of experimental study of strain state of elastically deformable solid.



Aim: calculate [q].

<u>Aim</u>: calculate d_1, d_2 .

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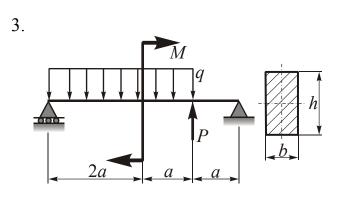
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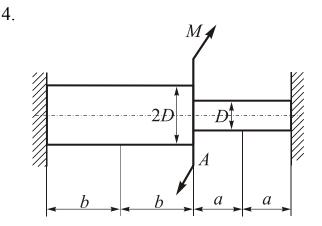
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 60

- 1. Experimental study of mechanical properties of structural materials. Description of the method and test device. Properties of material strength and ductility.
- 2. Method of the opening of static indeterminacy of the rod systems (consider an example and describe general procedures).



- <u>Given:</u> $a=1.0 \text{ m}, M=10 \text{ kNm}, q=20 \text{ kN/m}, b=5 \times 10^{-2} \text{ m}, h/b=2, [\sigma]=200 \text{ MPa}.$
- **<u>Aim</u>**: check the strength of the beam and draw the graphs $\sigma_x(z)$ and $\tau_{xz}(z)$ in corresponding critical cross-sections.



- <u>Given:</u> a=0.8 m, b=1.2 m, M=30 kNm, $[\tau]=70 MPa, G=8 \times 10^4 MPa.$
- <u>Aim:</u> calculate D and the angle of twist of A-section.

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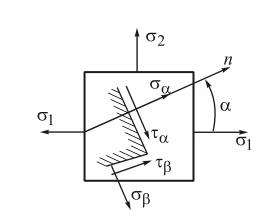
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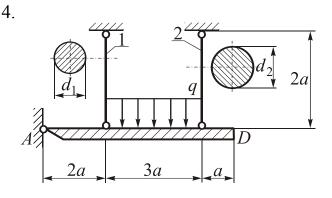
Examination card № 61

- 1. Transverse bending of prismatic bar. Proof of Juravsky formula for shear stresses in transverse bending. The graphs of shear stress distribution in round, rectangle and I-beam sections.
- 2. Compare the "load-displacement" and "stress-strain" diagrams for ductile material.



<u>Given:</u> $\sigma_1 = 100MPa$, $\sigma_2 = 60MPa$, $\alpha = 30^{\circ}$

<u>Aim</u>: calculate $\sigma_{\alpha}, \sigma_{\beta}, \tau_{\alpha}, \tau_{\beta}$.



<u>Given:</u> $a=1.0 m, d_1=4\times 10^{-2} m, d_{2=}6\times 10^{-2} m,$ $[\tau]=160 MPa, AD-absolutely rigid beam.$

<u>Aim</u>: calculate [q].

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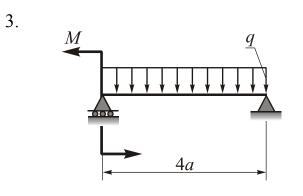
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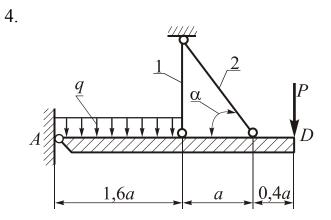
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 62

- 1. Method of opening of static indeterminacy of the shafts in torsion (use the example).
- 2. Allowable stress and the method of its determination. Correlation between $[\tau]$ and $[\sigma]$.



- **<u>Given:</u>** a=1 m, M=40kN/m, q=20kN/m, $[\sigma]=160 MPa$, Cross-section – I-beam N 20.
- Aim: check the strength.



<u>Given:</u> a=1 m, P=100 kN, q = 20kN/m, $d_1 = d_2 = d$, $\alpha = 60^\circ$, cross-section – round, $[\sigma] = 160 MPa$, AD – absolutely rigid beam.

Aim: calculate d.

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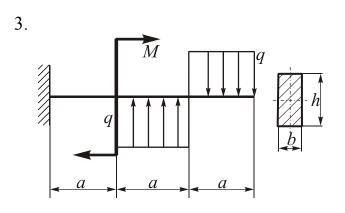
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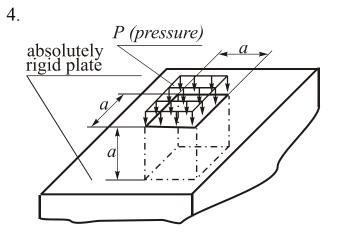
Examination card № 63

- 1. Types of bending. Bending formula for prismatic beam in pure plane bending (proof). Condition of strength. Stress distribution in round cross-section.
- 2. Poisson's ration and its determination.



<u>Given:</u> a=1 m, M=40 kNm, $[\sigma]=200 MPa$, q=20 kN/m, h/b=2.

<u>Aim</u>: calculate *h*, *b*, draw the graph of $\sigma_x(z)$ in critical section.



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<u>Given:</u> a=0.1 \times 10^{-2} m, P=100 MPa,
\mu=0.28, E=2 \times 10^{5} MPa.
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<u>Aim:</u> calculate stresses on the faces of absolutely elastic element and its relative change in volume.

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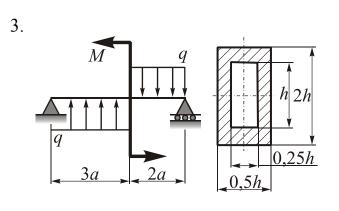
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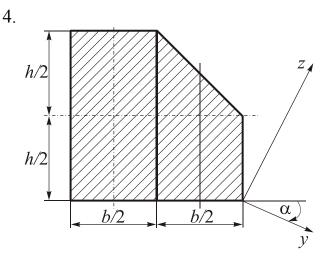
Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 64

- 1. Torsion of the shafts with rectangle cross-section. Critical points, calculation of maximum stresses, condition of strength.
- 2. Axial moments of inertia. Formula for calculation central axial moments of inertia of round, rectangle and right triangle sections. Correlation between axial and polar moments of inertia.



- <u>Given:</u> a=2 m, M=20 kNm, q=30 kN/m, $[\sigma]=200 MPa$, h=20 cm.
- Aim: check the strength of the beam.



- **<u>Given:</u>** $b=20\times 10^{-2}$ m, $h=20\times 10^{-2}$ m, $\alpha=30^{\circ}$.
- <u>Aim</u>: calculate axial moments of inertia and also product of inertia of composite cross-section relative to y, z axes.

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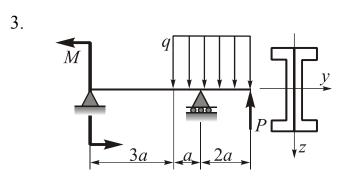
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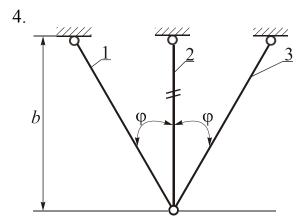
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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 65

- 1. Transverse plane bending of prismatic beam. Proof of Juravsky formula for shear stress distribution over the cross-section.
- 2. Principal hypotheses and assumptions in mechanics of material





- <u>Given:</u> a=1 m, q=10 kN/m, M=20 kNm, P=10 kN, $[\sigma]=180 MPa$.
- <u>Aim:</u> determine the number of channel section of composite beam.

Aim: calculate stresses in rods after assembly.

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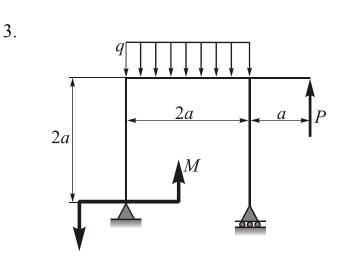
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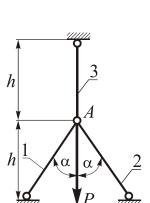
Examination card № 66

- 1. Differential relationships between q(x), $Q_z(x)$, $M_y(x)$ in plane bending (proof) and their application for checking the graphs of internal forces distribution (an example).
- 2. Hypothesis of plane sections and its use in proof of general formulae of tensioncompression, torsion, bending.

4.



- <u>Given:</u> a=0.5 m, M=6 kNm, q=18 kN/m, P=10 kN, $\lceil \sigma \rceil = 200 MPa$.
- <u>Aim:</u> draw graphs $N_x(x)$, $Q_z(x)$, $M_y(x)$; determine the diameter of round cross-section.



<u>Given:</u> $h=1 \text{ } M, \quad A_1=A_2=A, \quad A_3=2A,$ $\alpha=30^\circ, \quad [\sigma]=140 \text{ } MPa, \quad P=40 \text{ } kN,$ $E_1=E_2=E_3=E=2\times 10^5 \text{ } MPa$.

<u>Aim:</u> determine cross-sectional area *A* and vertical displacement of the hinge *A*.

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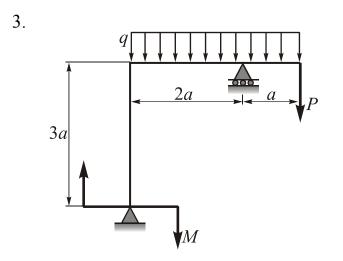
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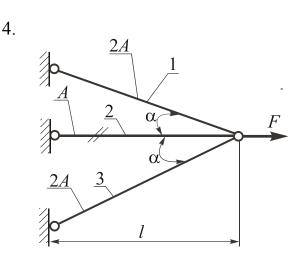
Examination card № 67

- 1. Generalized Hooke's law. Proof of its equations in terms of principal stresses.
- 2. Elastic moduli and their correlation



<u>Given:</u> a=1 m, M=6 kNm, q=10 kN/m, P=10 kN, $[\sigma]=160 MPa$.

<u>Aim:</u> draw graphs $N_x(x)$, $Q_z(x)$, $M_y(x)$; determine the diameter of the frame.



<u>Given:</u> $E=2\times10^5$ MPa, F=60 kN $A=2\times10^{-4}$ m², l=2 m, $\alpha=30$ °, $\Delta t = 50$ °K, $\alpha_t = 1.75\times10^{-5}$ 1/K **Aim:** determine stresses in the rods 1, 2, 3.

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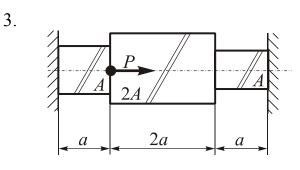
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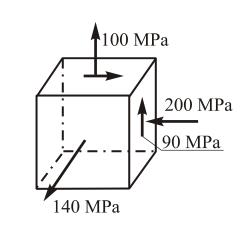
Examination card № 68

1. Proof of generalized Hooke's law equations.

2. Sectional moduli of cross-sections (definition and examples of calculation for different cross-sections).

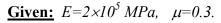
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<u>Given:</u> a=1 m, P=80 kN, $A=10cm^{2}$, $E=2\times 10^{11} Pa$, $\alpha_{t}=17.5\times 10^{-6} 1/K$, $\Delta t=40$ %.

Aim: calculate stresses in the rod.



<u>Aim:</u> determine position of principal planes and principal stresses and also relative change in volume ε_{ν} .

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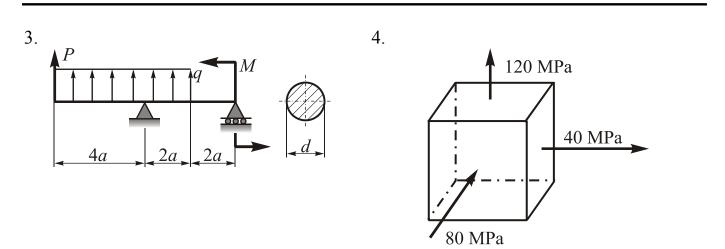
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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 69

- 1. Proof of generalized Hooke's law equations.
- 2. Method of sections as the principle of determination of internal forces.



<u>Given:</u> a=1 m, d=0.08 m, $M=2qa^2, P=2qa, [\sigma]=140 MPa.$

Aim: determine [q].

<u>Given:</u> $\mu = 0.28$, $E = 2 \cdot 10^{11} Pa$.

<u>Aim:</u> determine deformations of element's edges and relative change of its volume.

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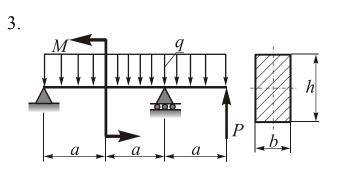
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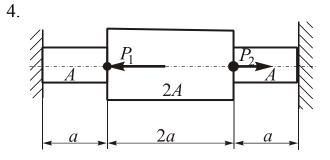
Examination card № 70

- 1. Torsional deformation of solid round shafts. Formula for angle of twist (proof). Condition of rigidity.
- 2. Generalized Hooke's law and its application for calculation of stresses, strains and relative change of elastic material volume.



<u>Given:</u> a=1 m, M=40 kNM, q=20 kN/m, P=10 kN, h/b=2, $[\sigma]=160 MPa$.

Aim: calculate b, h.



<u>Given:</u> a=0.1 m, $P_1=40 kN$, $P_2=10 kN$, $[\sigma]=160 MPa$.

<u>Aim:</u> determine internal forces N_x , calculate cross-sectional area A, determine acting stresses σ_x .

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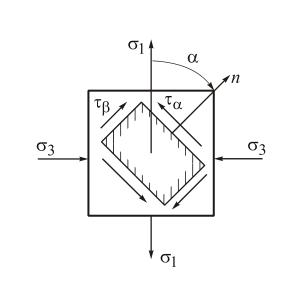
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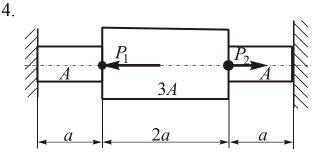
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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 71

- 1. Torsional deformation of solid round shafts. Formula for angle of twist (proof). Condition of rigidity.
- 2. Generalized Hooke's law and its application for calculation of stresses, strains and relative change of elastic material volume.





<u>Given:</u> $\sigma_1 = 150 MPa$, $\sigma_3 = 100 MPa$, $\alpha = 45^{\circ}$

<u>Aim</u>: calculate σ_{α} , τ_{α} , σ_{β} , τ_{β} .

- **<u>Given:</u>** $a=1.0 \text{ m}, P_1=40 \text{ kN}, P_2=30 \text{ kN}, [\sigma]=160 \text{ MPa}.$
- **<u>Aim:</u>** determine internal forces N_x , calculate cross-sectional area A, determine acting stresses σ_x .

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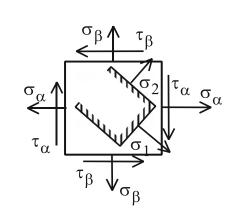
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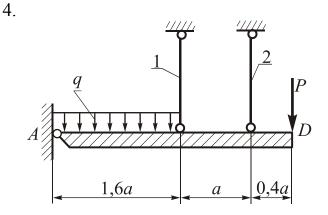
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Degree B. Sc. Branch of education: 1001 Aerospace Engineering Semester III Course "Mechanics of materials"

Examination card № 72

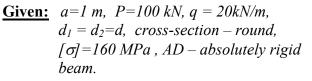
- 1. Torsional deformation of solid round shafts. Formula for angle of twist (proof). Condition of rigidity.
- 2. Generalized Hooke's law and its application for calculation of stresses, strains and relative change of elastic material volume.





<u>Given:</u> $\sigma_{\alpha} = 100 MPa$, $\tau_{\alpha} = 60 MPa$, $\sigma_{\beta} = 50 MPa$

<u>Aim</u>: calculate σ_1 , $\sigma_{2(3)}$, α_p .



Aim: calculate d.

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